



# Standard Test Method for Measurement of Extreme-Pressure Properties of Lubricating Fluids (Timken Method)<sup>1</sup>

This standard is issued under the fixed designation D2782; the number immediately following the designation indicates the year of original adoption or, in the case of revision, the year of last revision. A number in parentheses indicates the year of last reapproval. A superscript epsilon ( $\epsilon$ ) indicates an editorial change since the last revision or reapproval.

## 1. Scope

1.1 This test method covers the determination of the load-carrying capacity of lubricating fluids by means of the Timken Extreme Pressure Tester.

NOTE 1—This test method is suitable for testing fluids having a viscosity of less than about 5000 cSt (5000 mm<sup>2</sup>/s) at 40°C. For testing fluids having a higher viscosity, refer to [Note 5](#) in [9.1](#).

1.2 The values stated in SI units are to be regarded as standard. Because the equipment used in this test method is available only in inch-pound units, SI units are omitted when referring to the equipment and the test specimens.

1.3 *This standard does not purport to address all of the safety concerns, if any, associated with its use. It is the responsibility of the user of this standard to establish appropriate safety and health practices and determine the applicability of regulatory limitations prior to use.* Specific warning statements are given in [7.1](#), [7.2](#), [8.1](#), [8.2](#), [9.4](#), and [9.9](#).

## 2. Referenced Documents

### 2.1 ASTM Standards:<sup>2</sup>

[D2509](#) Test Method for Measurement of Load-Carrying Capacity of Lubricating Grease (Timken Method)

[D4175](#) Terminology Relating to Petroleum, Petroleum Products, and Lubricants

[G40](#) Terminology Relating to Wear and Erosion

<sup>1</sup> This test method is under the jurisdiction of ASTM Committee D02 on Petroleum Products and Lubricants and is the direct responsibility of Subcommittee D02.L0.11 on Tribological Properties of Industrial Fluids and Lubricants.

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<sup>2</sup> For referenced ASTM standards, visit the ASTM website, [www.astm.org](http://www.astm.org), or contact ASTM Customer Service at [service@astm.org](mailto:service@astm.org). For *Annual Book of ASTM Standards* volume information, refer to the standard's Document Summary page on the ASTM website.

### 2.2 ASTM Adjuncts:

Photograph of Test Blocks Showing Scars<sup>3</sup>

## 3. Terminology

### 3.1 Definitions:

3.1.1 *extreme pressure (EP) additive, n, in a lubricant*—a substance that minimizes damage to metal surfaces in contact under high-stress rubbing conditions. **D4175**

3.1.2 *lubricant, n*—any substance interposed between two surfaces for the purpose of reducing friction or wear between them. **G40**

3.1.3 *scoring, n, in tribology*—a severe form of wear characterized by the formation of extensive grooves and scratches in the direction of sliding. **G40**

3.1.4 *wear, n*—damage to a solid surface generally involving progressive loss of material, due to relative motion between that surface and a contacting substance or substances. **G40**

### 3.2 Definitions of Terms Specific to This Standard:

3.2.1 *load-carrying capacity of a lubricant*—as determined by this test method, the maximum load or pressure that can be sustained by the lubricant (when used in the given system under specific conditions) without failure of the sliding contact surfaces as evidenced by scoring or seizure or asperity welding.

3.2.2 *OK value, n*—as determined by this test method, the maximum mass (weight) added to the load lever weight pan, at which no scoring or seizure occurs.

3.2.3 *score value, n*—as determined by this test method, the minimum mass (weight) added to the load lever weight pan, at which scoring or seizure occurs.

3.2.3.1 *Discussion*—When the lubricant film is substantially maintained, a smooth scar is obtained on the test block, but when there is a breakdown of the lubricant film, scoring or surface failure of the test block takes place, as shown in [Figs. 1 and 2](#). In its simplest and most recognized form, scoring is

<sup>3</sup> Available from ASTM International Headquarters. Order Adjunct No. [ADJD2509](#). Original adjunct produced in 1972.

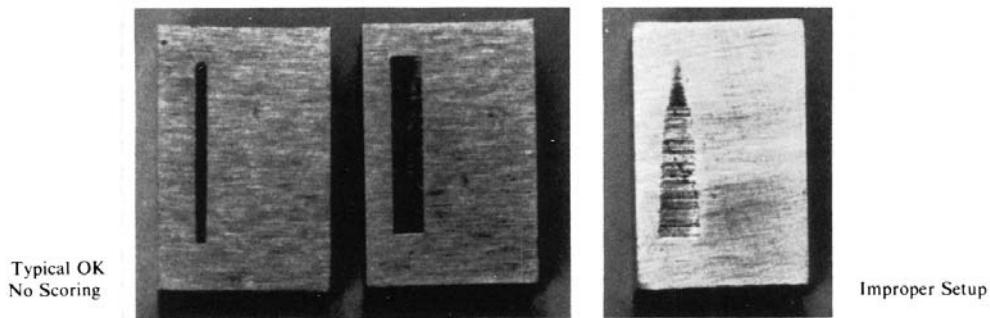


FIG. 1 Test Blocks Showing Various Types of Scar

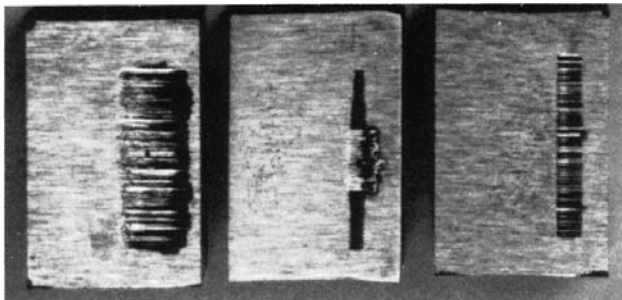


FIG. 2 Scoring

characterized by the furrowed appearance of a wide scar on the test block and by excessive pick-up of metal on the surface of the test cup. The form of surface failure more usually encountered, however, consists of a comparatively smooth scar, which shows local damage that usually extends beyond the width of the scar. Scratches or striations that occur in an otherwise smooth scar and that do not extend beyond the width of the scar are not considered as evidence of scoring.

3.2.4 *seizure or asperity welding*—localized fusion of metal between the rubbing surfaces of the test pieces. Seizure is usually indicated by streaks appearing on the surface of the test cup, an increase in friction and wear, or unusual noise and vibration. Throughout this test method the term *seizure* is understood to mean *seizure or asperity welding*.

#### 4. Summary of Test Method

4.1 The tester is operated with a steel test cup rotating against a steel test block. The rotating speed is  $123.71 \pm 0.77$  m/min ( $405.88 \pm 2.54$  ft/min) which is equivalent to a spindle speed of  $800 \pm 5$  rpm. Fluid samples are preheated to  $37.8 \pm 2.8^\circ\text{C}$  ( $100 \pm 5^\circ\text{F}$ ) before starting the test.

4.2 Two determinations are made: the minimum load (score value) that will rupture the lubricant film being tested between the rotating cup and the stationary block and cause scoring or seizure; and the maximum load (OK value) at which the rotating cup will not rupture the lubricant film and cause scoring or seizure between the rotating cup and the stationary block.

#### 5. Significance and Use

5.1 This test method is used widely for the determination of extreme pressure properties for specification purposes. Users

are cautioned to carefully consider the precision and bias statements herein when establishing specification limits.

#### 6. Apparatus

6.1 *Timken Extreme Pressure Tester*,<sup>4</sup> described in detail in **Annex A1** and illustrated in **Fig. 3**.

6.2 *Sample Feed Device*,<sup>4</sup> for supplying the test specimens with fluid is described in **Annex A1**.

6.3 *Loading Mechanism*,<sup>4</sup> for applying and removing the load weights without shock at the uniform rate of 0.91 to 1.36 kg/s (2 to 3 lb/s). A detailed description is given in **Annex A1**.

6.4 *Microscope*,<sup>4</sup> low-power (50× to 60×) having sufficient clearance under objective to accommodate the test block. It should be fitted with a filar micrometer so that the scar width may be measured with an accuracy of  $\pm 0.05$  mm ( $\pm 0.002$  in.).

6.5 *Timer*, graduated in minutes and seconds.

#### 7. Reagents and Materials

7.1 *Acetone*, reagent grade. (**Warning**—Extremely flammable. Harmful when inhaled. See **A3.1**.)

7.2 *Stoddard Solvent or White Spirit*, reagent grade. (**Warning**—Flammable. See **A3.2**.)

7.3 *Test Cup*,<sup>4,5</sup> of carburized steel, having a Rockwell Hardness “C” Scale Number of 58 to 62, or a Vickers Hardness Number of 653 to 756. The cups have a width of  $0.514 \pm 0.002$  in., a perimeter of  $6.083 \pm 0.009$  in., a diameter of  $1.938 + 0.001, - 0.005$  in. and a maximum radial run-out of 0.0005 in. The axial surface roughness should lie between 0.51 and  $0.76 \mu\text{m}$  (20 and 30  $\mu\text{in.}$ ) C.L.A.

7.4 *Test Blocks*,<sup>4,6</sup> with test surfaces  $0.485 \pm 0.002$  in. wide and  $0.750 \pm 0.016$  in. long, of carburized steel, having a Rockwell Hardness “C” Scale Number of 58 to 62, or a Vickers Hardness Number of 653 to 756. Each block is supplied with four ground faces and the surface roughness should lie between 0.51 and  $0.76 \mu\text{m}$  (20 and 30  $\mu\text{in.}$ ) C.L.A.

#### 8. Preparation of Apparatus

8.1 Clean the apparatus with (1) Stoddard solvent or White Spirit, and (2) acetone and blow dry. (**Warning**—Extremely

<sup>4</sup> The sole source of supply of the apparatus known to the committee at this time is Falex Corp., 1020 Airpark Dr., Sugar Grove, IL 60554. If you are aware of alternative suppliers, please provide this information to ASTM International Headquarters. Your comments will receive careful consideration at a meeting of the responsible technical committee,<sup>1</sup> which you may attend.

<sup>5</sup> Available from Falex Corp., under Part No. F-25061.

<sup>6</sup> Available from Falex Corp., under Part No. F-25001.

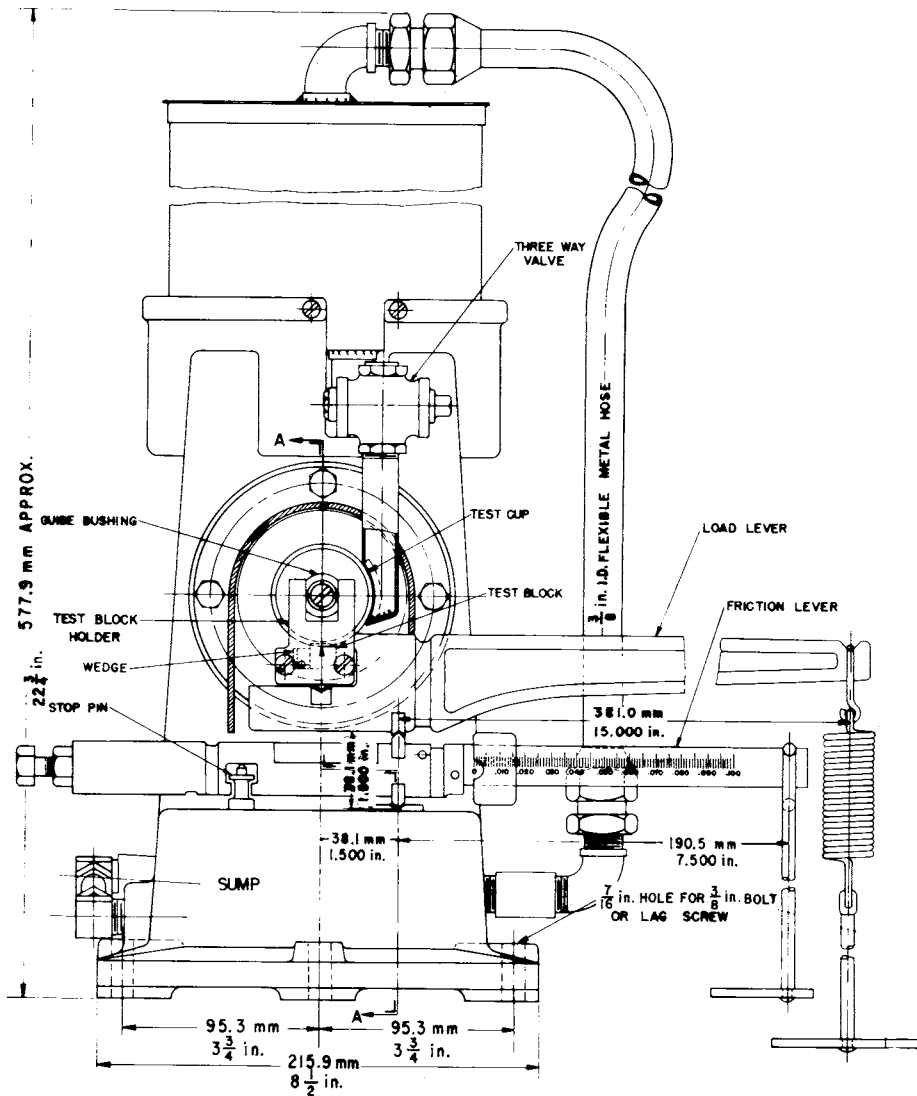


FIG. 3 Timken Tester

flammable. Harmful when inhaled. See A3.1.) (**Warning**—Flammable. See A3.2.) Flush with approximately 1 L (1 qt) of the fluid to be tested. Discard the flushing fluid. (**Warning**—Since acetone is highly flammable, use the minimum quantity.)

8.2 Select a new test cup and block, wash with Stoddard solvent or White Spirit (**Warning**—Flammable. See A3.2.) and dry with a clean soft cloth or paper. Immediately before use rinse the test cup and block with acetone and blow them dry. Do not use solvents such as carbon tetrachloride or others that may inherently possess load-carrying properties which may effect the results.

NOTE 2—This cleaning may be done in an ultrasonic cleaner.

8.3 Assemble the tester carefully (Fig. 4), placing the test cup on the spindle and making certain that it is well seated, drawing it up firmly but avoiding possible distortion from excessive tightening (Note 3). Place the test block in the test block holder and adjust the levers so that all the knife edges are in proper alignment. Exercise special care in placing the stirrup of the spring-weight platform assembly (selection of which will depend on the loading device) in the groove of the

load-lever arm to avoid premature shock to the test block when the load is applied. To ensure that the test block, test block holder, and lever arms are properly aligned and seated, coat the test block and test cup with the lubricant to be tested, and rotate the machine slowly for a few revolutions either by hand or by suitable control mechanism. When the parts are in alignment, the fluid will be wiped off the cup over its entire width.

NOTE 3—At this point it is recommended that a dial indicator used to check the radial run-out of the cup *in situ* not exceed 0.025 mm (0.001 in.) total indicator movement.

## 9. Procedure

9.1 Fill the reservoir of the tester to within 76 mm (3 in.) of the top (approximately 3 L or 3 qt) with the fluid to be tested. Preheat the fluid to  $37.8 \pm 2.8^\circ\text{C}$  ( $100 \pm 5^\circ\text{F}$ ).

NOTE 4—The fluid may be heated by the use of an immersion heater located in the tester reservoir or by heating the fluid prior to filling the reservoir. If an immersion heater is used, localized overheating must be avoided. This may be done by stirring or by circulating prior to the assembly of the lever arm.

NOTE 5—Fluids having a viscosity above about 5000 cSt (5000 mm<sup>2</sup>/s)

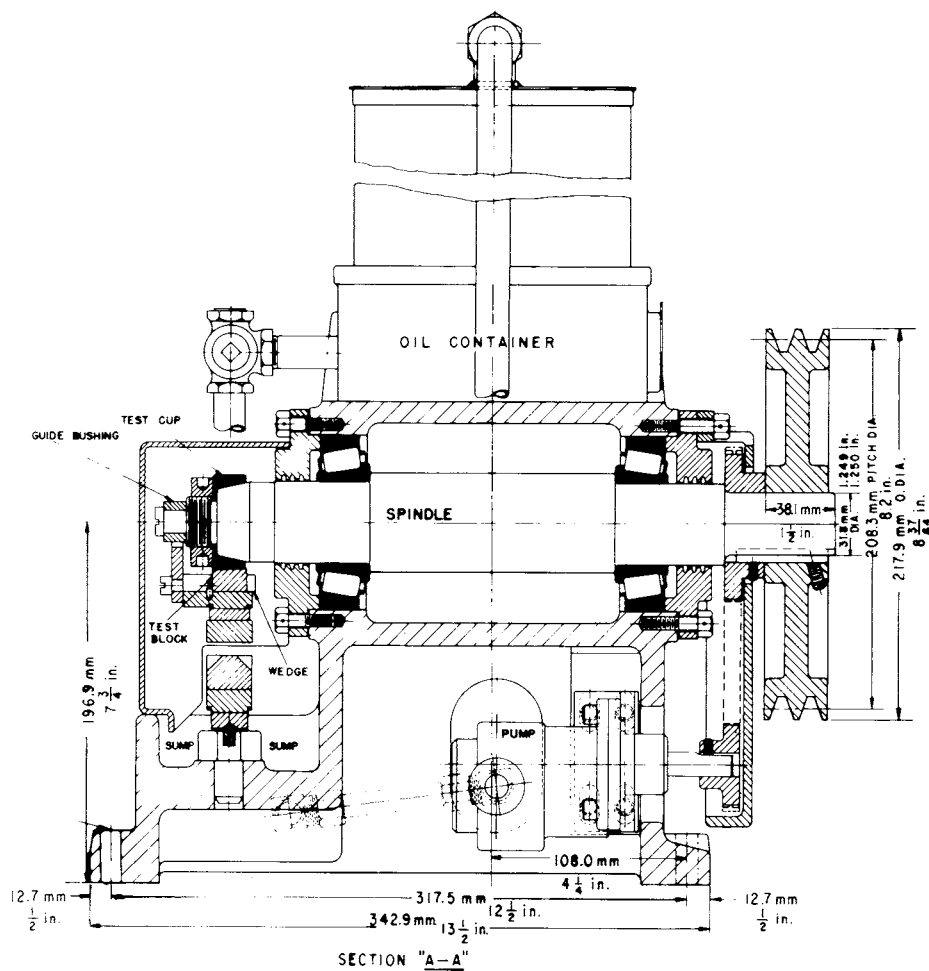


FIG. 3 (continued)

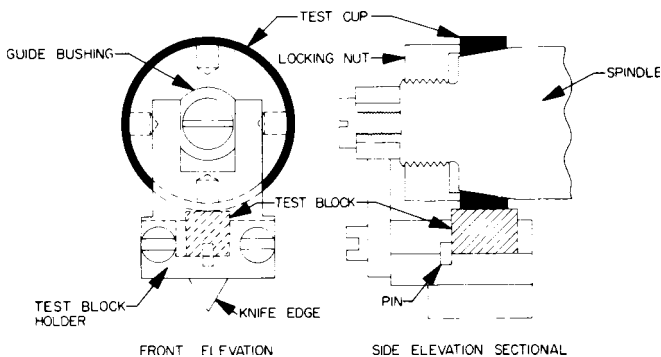


FIG. 4 Assembly of Tester Showing Test Pieces

at 40°C often cannot be tested at the prescribed fluid temperature of 37.8 ± 2.8°C (100 ± 5°F) because of inability of the pump to recirculate the fluid at this temperature. However, results from limited cooperative tests, covered in Tables A1.1 and A1.2, indicate that the starting fluid temperature could be increased to 65.6°C (150°F) to obtain adequate flow without affecting OK or score values. Testing of such high-viscosity fluids at room temperature in the Test Method D2509 grease feeder also appears valid but may be difficult because of fluid leakage.

9.2 Set the discharge valve at full open. Allow the lubricant to flood the test cup and block. When the sump is about half filled with the fluid, start the motor and run for 30 s to break-in.

If the equipment used is equipped with acceleration control, start the motor and increase the spindle speed gradually to achieve 800 ± 5 rpm after 15 s. Run for a further 15 s to complete the break-in.

9.3 After a break-in period of 30 s, start the timer and apply at 8.9 to 13.3 N/s (2 to 3 lbf/s), a load that is less than the expected score load. In the absence of a better estimate, a starting load of 30 lbf is recommended. The load-lever arm, spring-weight platform assembly is not considered a part of the applied load. In the event a lower starting load is used, it must be a multiple of 6. Then allow the machine to run at 800 ± 5 rpm for 10 min ± 15 s after load application is initiated, unless a score is detected before that period.

9.4 If, after the load has been applied, scoring is evident by vibration or noise, stop the tester at once, turn off the flow of lubricant, and remove the load. Since the excessive heat developed with deep scoring may alter the surface characteristics of the entire block, discard the test block. (Warning—The machine and test pieces may be hot at this point and care should be exercised in their handling.)

9.5 If no scoring is detected, allow the tester to run for 10 min ± 15 s from the start of the application of the load. At the end of the 10-min ± 15-s period, reverse the loading device and remove the load from the lever arm. Turn off the motor,

allow the spindle to come to rest, then turn off the flow of fluid. Remove the load lever and inspect the condition of the test block surface at 1× magnification. Microscopical observations shall not be used to define when scoring has occurred. The lubricant has failed at the imposed load if the wear scar indicates any scoring or welding.

NOTE 6—A microscope may be used to examine the wear scar for further information as required in 9.9.

9.6 If no score is observed, turn the test block to expose a new surface of contact and, with a new test cup, repeat the test, as in 9.5, at 10-lbf increments until a load that produces a score is reached. At this point decrease the load by 5 lbf for the final determination.

NOTE 7—Before each test in 9.6-9.8 cool the fluid in the reservoir to  $37.8 \pm 2.8^\circ\text{C}$  ( $100 \pm 5^\circ\text{F}$ ), cool the shaft to less than  $65.6^\circ\text{C}$  ( $150^\circ\text{F}$ ), install a new test cup, and turn the test block to expose a new surface of contact. When seizure has occurred, discard the entire test block since excessive heat, developed when scarring occurs, may alter the surface characteristics of the entire block.

9.7 If a score is produced at the 30-lbf load, reduce the load by 6-lbf decrements until no scoring is realized. At this point, increase the load by 3 lbf for the final determination.

9.8 When the wear scar evidence at any load stage makes the definition of the onset of scoring questionable, repeat the test at the same load. If the second test produces a score, record a score rating for this load. Similarly, if the second test produces no scoring, record a no score rating. If the second test again yields a questionable result, simply withhold judgment of the rating at this load stage and test the fluid at the immediately next higher load stage (see Annex A2). Then assign a rating to the load stage in question that is identical to the rating obtained at the immediately next higher load stage employed.

9.9 After the OK value has been determined, remove the test block and wash with Stoddard solvent or White Spirit, rinse with acetone (**Warning**—Extremely flammable. Harmful when inhaled. See A3.1.) (**Warning**—Flammable. See A3.2.), and blow dry. By means of a filar micrometer microscope, measure the width of the scars on those blocks which successfully carried this load. Make all measurements to 0.05 mm (0.002 in.).

## 10. Calculation

10.1 When desired, the contact (unit) pressure that exists between the cup and block at the conclusion of the test can be calculated. Calculate the contact pressure,  $C$ , as follows:

$$C, \text{ psi} = [L(X + G)]/YZ \text{ or } [20(X + G)]/Z \quad (1)$$

$$C, \text{ MPa} = 9.81[L(X' + 0.454G)]/Y'Z' \quad (2)$$

where:

$L$  = mechanical advantage of load-lever arm, 10,

$G$  = load-lever constant (value is stamped on lever arm of each tester),

$X$  = mass (weight) placed on the weight pan, lb,

$X'$  = mass (weight) placed on the weight pan, kg,

$Y$  = length of test scar ( $\frac{1}{2}$  in.),

$Y'$  = length of test scar (12.7 mm),

$Z$  = average width of test scar, in., and

$Z'$  = average width of test scar, mm.

10.2 For convenience, contact (unit) pressures in pounds per square inch are listed in Table X3.1.

## 11. Report

11.1 Report the OK and score values in terms of the mass (weights) placed on the weight pan hanging from the end of the load-lever arm; do not include the mass (weight) of the pan assembly. Report the values in multiples of 5 lb above 30 lb and in multiples of 3 lb below 30 lb.

## 12. Precision and Bias <sup>7</sup>

12.1 The precision and this test method as determined by the statistical examination of interlaboratory test results is as follows:

12.1.1 *Repeatability*—The difference between successive results obtained by the same operator with the same apparatus under constant operating conditions on identical test material would, in the long run, in the normal and correct operation of the test method exceed the following values only in one case in twenty.

Repeatability = 30 % of the mean value

12.1.2 *Reproducibility*—The difference between two single and independent results obtained by different operators working in different laboratories on identical test material would, in the long run, exceed the following values only in one case in twenty.

Reproducibility = 74 % of the mean value

NOTE 8—Precision data were obtained from round-robin tests by eleven laboratories on seven paraffinic base oil blends.<sup>7</sup> A table of raw data from the round robin is provided in Appendix X1 for information only.

NOTE 9—The precision values given in Section 12 are considered applicable for samples having, in the long run, an average Timken OK load of 15 lbf minimum.

NOTE 10—The following equipment, as listed in D02-1223,<sup>7</sup> was used to develop the precision statement and no statistically significant differences were found between these pieces of equipment: (1) Falex Timken EP Tester, 1020 Airpark Drive, Sugar Grove, IL 60555. (2) The Timken Company, Canton, OH. To date, no other equipment has demonstrated through ASTM interlaboratory testing the ability to meet the precision of this test. This is not an endorsement or certification by ASTM.

## 13. Keywords

13.1 EP; extreme pressure; load carrying capacity; lubricant; Timken

<sup>7</sup> Supporting data have been filed at ASTM International Headquarters and may be obtained by requesting Research Report D02-1223.

ANNEXES

(Mandatory Information)

A1. TIMKEN EXTREME PRESSURE TESTER

A1.1 *Timken Extreme Pressure Tester*,<sup>4</sup> consisting essentially of a steel test cup rotating against a steel test block loaded from below. The test cup is attached to a horizontal spindle mounted in two roller bearings and driven at 800 ± 5 rpm by a 2-hp (1.5-W) synchronous motor. The test block is mounted in a holder upon knife-edge bearings, designed to promote correct alignment and uniform pressure between the test cup and block. The machine must be mounted rigidly as results are affected by vibration. See **Table A1.1** and **Table A1.2** for test results.

A1.1.1 *Test Block Holder*, fitted with a pin, is provided with a steel wedge to hold the test block in position. It also has a pair of arms which fit around a cast iron guide bushing on the spindle. The bottom of the holder is mounted on knife edges on the load lever.

A1.1.2 *Test Cup Spindle*, tapered to receive the test cup which is locked in position by a locking nut with a left-hand thread. The spindle has a maximum radial run out of 0.013 mm (0.0005 in.); if the assembled cup and spindle has a radial run out greater than 0.025 mm (0.001 in.), test results can be affected. This value would indicate a badly worn or damaged spindle which should be replaced. Periodic checking of an assembled cup and spindle is recommended.

A1.1.3 *Lever System*, consisting of two levers: the upper or load lever and the bottom or friction lever. The load lever carries the test block holder and is mounted on knife edges on the friction lever. The friction lever, pivoted on a knife edge, is provided with a stop at the unloaded end.

A1.1.4 *Load-Lever Constant*—The mechanical advantage of the load lever is 10; that is, 1 lb placed on the notch at the outer end will exert a force of 10 lbf on the test block. The effective weight of the load-lever arm and weight pan system is stamped on the lever arm of each tester.

A1.1.5 *Sample Feed Device*—A 3785-mL (1-gal) capacity reservoir and piping allows gravity flow of test fluid over test cup and block. The reservoir is fitted with an electric heater so that test fluid may be heated. The test fluid flows into a sump and is removed by a pump which returns the fluid to the reservoir. A 100-mesh screen placed in the sump outlet prevents wear particles from entering the fluid system. A magnet placed in the sump outlet is also suggested for this purpose. Flow rate of fluid onto the test cup and block is controlled by a three-way valve at the reservoir outlet.

A1.2 *Loading Mechanism*<sup>4</sup>, consists of a power-operated loading platform so arranged that the weights are applied to the end of the load lever at a uniform rate of 0.91 to 1.36 kg/s (2 to 3 lb/s), thus eliminating any errors due to a non-uniformity of load application. The weights are applied vertically to the center of the pan at the end of the load lever. Note that the loading rate is a function of the velocity of the loading mechanism and the deflection rate of the weight carrier springs. To measure the loading rate of the mechanism, the following procedure can be used.

A1.2.1 Place a piece of paper on the loading platform. Over it place the weight pan, with a 10 or 20-lb (4.54 or 9.07-kg) mass (weight) on the pan. An edge of the paper must be left exposed.

A1.2.2 Start the loading platform. When loading begins (indicated by loss of slack in the pan suspension apparatus) begin timing using a stopwatch.

A1.2.3 Grip the paper under the weight pan firmly. When the paper slides out from between the pan and platform, the stopwatch must be stopped. The time elapsed is the time to apply the load on the pan.

A1.2.4 Repeat **A1.2.1**, **A1.2.2**, and **A1.2.3** at 10 or 20-lbf increments through the maximum load to be used on the tester (smaller increments may be necessary if the loading spring is very non-linear).

A1.2.5 Plot corresponding load versus time values and draw a curve through them. The slope at all points should be between 8.9 to 13.3 N/s (2 and 3 lbf/s). Alternatively, the loading rate for each load increment can be calculated as illustrated as follows, for a 20-lbf (89-N) increment between 20 and 40 lbf (89 and 177.9 N).

Load lbf = N	Time to Apply, s
0	0
20	7.8
40	14.6
60	21.3

Rate

$$(20 \text{ to } 40 \text{ lbf}) = \frac{40 \text{ lbf} - 20 \text{ lbf}}{14.6 \text{ s} - 7.8 \text{ s}} = \frac{20 \text{ lbf}}{6.8 \text{ s}} = 2.9 \text{ lbf/s} \quad (\text{A1.1})$$

$$(89 \text{ to } 177.9 \text{ N}) = \frac{177.9 \text{ N} - 89 \text{ N}}{14.6 \text{ s} - 7.8 \text{ s}} = \frac{88.9 \text{ N}}{6.8 \text{ s}} = 13.07 \text{ N/s} \quad (\text{A1.2})$$

**TABLE A1.1 Results of 4-Machine, 3-Laboratory Study of Flow Rates of High-Viscosity Fluids in Timken Tester**

Fluid Viscosity, cSt (mm <sup>2</sup> /s) at 37.8°C	Lowest	Highest
214	1680	2250
648	593	794
1519	256	321
2202	162	209

<sup>4</sup> Calculated from the time for delivery of 900 mL of fluid from full reservoir.

**TABLE A1.2 Viscosities and Estimated Flow Rates for Two Fluids (Low and High Viscosity)**

	Viscosity, cSt SUS			Estimated Flow Rate, <sup>A</sup> mL/min, D2782, Valve Full Open			
	37.8°C (100°F)	52.5°C (125°F) (ext.)	65.6°C (150°F) (ext.)	99°C (210°F)	37.8°C (100°F)	52.5°C (125°F)	65.6°C (150°F)
Fluid A	218.5 (1 012)	103.1 (470)	53.8 (250)	18.44 (86)	1650–2300	3500–4500	6500–8000
Fluid B	5 505 (25 500)	1 681 (7 800)	624 (2 900)	105.5 (492)	70–90	240–300	570–800

<sup>A</sup> Estimates from plot of temperature versus flow rate data on the four fluids shown in Table A1.1.

All increments must show values in the range from 98.9 to 13.3 N/s (2.0 to 3.0 lbf/s).

A1.2.6 Once the loading rate has been established, it can be adjusted, if necessary, by either changing the platform descent

rate, or by switching to a spring having different elongation behavior when loaded.

## A2. PROCEDURE FOR ASSIGNING RATING IN CASE OF QUESTIONABLE EVIDENCE OF SCORING

A2.1 The procedure to be followed in the assignment of a score or non-score rating to a load stage at which the evidence of the onset of scoring is questionable is illustrated by considering the following examples. In each example it is assumed that in accordance with 9.8 duplicate results have previously been obtained at the load stage in question, and that the examination of wear scars so produced made the assignment of either score or no-score ratings uncertain.

A2.1.1 If a fluid is tested in duplicate at a load of 40 lbf (177.9 N) in accordance with 9.8, and the examination of both wear scars produced leaves some question regarding the onset of scoring, the operator must next test the fluid at a load of 45 lbf (200.2 N). If no score is observed at this higher load, a no-score rating will be entered for the 40 lbf (177.9 N) load, and testing will proceed at the next usual load increment, for example, 50 lbf (222.4 N). If a score is observed at the 45-lbf

(200.2-N) load, a score rating will be entered for the 40-lbf (177.9-N) load. In this case in accordance with 9.6 the next and final test would be conducted at the 35-lbf (155.7-N) load.

A2.1.2 If a fluid is tested in duplicate at a load of 24 lbf (106.8 N) in accordance with 9.8, and the examination of both wear scars produced leaves some question regarding the onset of scoring, the operator should next test the fluid at a load of 27 lbf (120.1 N). If a score is observed at this higher load, a score rating will be entered for the 24-lbf (106.8-N) load, and testing will proceed at the next usual load decrement, for example, 18 lbf (80.1 N). If no score is observed at the 27-lbf (120.1-N) load, a no-score rating will be entered for the 24-lbf (106.8-N) load. In this case in accordance with 9.7 no further testing is required since no-score ratings are recorded for the 24 and 27-lbf (106.8 and 120.1-N) loads and scoring had presumably previously been observed at the 30-lbf (133.4-N) load.

## A3. PRECAUTIONARY STATEMENTS

### A3.1 Acetone

Harmful if inhaled. Vapors may cause flash fire.

Keep away from heat, sparks, and open flame.

Keep container closed.

Use with adequate ventilation.

Avoid buildup of vapors and eliminate all sources of ignition, especially nonexplosion-proof electrical apparatus and heater.

Avoid prolonged breathing of vapor or spray mist.

Avoid prolonged or repeated skin contact.

### A3.2 Stoddard Solvent or White Spirit

Vapor may cause flash fire.

Keep away from heat, sparks, and open flame.

Keep container closed.

Use with adequate ventilation.

Avoid buildup of vapors and eliminate all sources of ignition, especially nonexplosion-proof electrical devices and heaters.

Avoid prolonged breathing of vapor or spray mist.

Avoid prolonged or repeated skin contact.

**APPENDIXES**
**(Nonmandatory Information)**
**X1. EXAMPLES OF TYPICAL TIMKEN OK LOADS**

X1.1 Shown in **Tables X1.1 and X1.2** are Timken OK values for samples studied in an interlaboratory test program to determine the current test method precision. A pair of results is shown by each laboratory for each sample. According to instructions for the interlaboratory round robin, replicate results were obtained by the same operator with the same apparatus under constant operating conditions during a short period of time. This data is provided solely as an illustration of the values that might be obtained within and between laboratories on various lubricating oil samples in the normal use of this test method.

**TABLE X1.1 Raw Data from 1985 Interlaboratory Study to Determine Test Method Precision for Test Method D2782**

NOTE 1—For Sample 102, several values are reported as multiples of 5 lb. This illustrates improper reporting or determination of the OK load. (See 9.7 and 10.1.)

NOTE 2—For Sample 106, several values were reported as “100 + lb.” In these cases, cooperators indicated they were unable to determine the fail load due to lack of sufficient weights or by choice did not exceed 100 lb on the lever arm.

Sample		Timken OK Load, lb						
Laboratory	Run	101	102	103	104	105	106	107
1	1	75	6	45	65	80	100+	21
	2	80	6	35	80	75	100+	35
2	1	65	5	30	65	45	85	30
	2	35	9	30	65	45	85	30
3	1	65	9	24	75	95	85	50
	2	60	6	18	75	80	110	50
4	1	65	9	55	80	60	85	55
	2	70	9	35	70	70	80	50
5	1	70	3	9	80	85	80	15
	2	30	3	15	90	85	85	15
6	1	65	10	60	70	70	100+	55
	2	65	10	55	65	85	100+	45
7	1	40	9	30	50	35	100+	50
	2	45	9	45	45	35	100+	45
8	1	75	5	35	60	70	100+	40
	2	75	5	30	60	75	100+	40
9	1	85	10	50	65	55	100	25
	2	75	10	40	70	65	100	45
10	1	70	12	12	60	65	85	40
	2	70	9	9	55	50	90	50
11	1	65	6	12	60	45	75	45
	2	70	6	15	60	50	75	35
Average		64	8	31	67	65	92	39



**TABLE X1.2 Raw Data from 1998 Mini Interlaboratory Study**

NOTE 1—With 7 cooperators and one test lubricant to study variability of Timken results on a Typical ISO 220 VG Sulfur/Phosphorus Industrial Gear Oil (Lubrizol OS#92133F).

Laboratory	Machine	Run No.	O.K Load, lb	Unit Load <sup>A</sup> , psi	Avg Scar Diam, mm	Avg Scar Diam, in
1	1	1	65	36 620	0.90	0.0355
		2	60	33 803	0.90	0.0355
		3	60	36 286	0.84	0.0331
		4	65	37 954	0.87	0.0343
	2	1	60	30 763	0.99	0.0390
		2	60	32 750	0.93	0.0366
		3	60	31 423	0.97	0.0382
		4	60	30 178	1.01	0.0398
2	1	1	70			0.0000
	2	1	65			0.0000
3	1	1	65	27 214	1.21	0.0478
		2	75	25 571	1.49	0.0587
4	1	1	65	29 535	1.12	0.0440
		2	65	29 195	1.13	0.0445
5	1	1	55	24 573	1.14	0.0448
		2	55	27 021	1.03	0.0407
6	1	1	65	29 430	1.12	0.0442
		2	65	27 748	1.19	0.0469
7	1	1	60	27 460		0.0437
		2	60	23 346		0.0514
	2	1	60	30 303		0.0396
		2	60	31 250		0.0384

<sup>A</sup> Calculated using formula: Unit Load (psi) = [20 × O.K. Load(lb)]/avg.scar. (in)]

## X2. EXAMPLES OF USE OF THE PRECISION STATEMENT

**X2.1 Repeatability**—Two determinations of the Timken OK load are made within a short period of time by the same operator on identical samples of Oil A. The same equipment is used for both determinations. Result 1 = 50 lb. Result 2 = 65 lb. Are these results within the repeatability of the test method?

$$\begin{aligned}
 &\text{Result 1} = 50 \text{ lb} \\
 &\text{Result 2} = 65 \text{ lb} \\
 X &= \text{Average of the two results} = 58 \text{ lb} \\
 Y &= \text{Repeatability limits for the two results} \\
 &= 30 \% \text{ of the mean value } X \\
 &= 0.30 \times 58 \text{ lb} \\
 &= 17 \text{ lb} \\
 Z &= \text{Difference between results} \\
 &= 65 \text{ lb} - 50 \text{ lb} \\
 &= 15 \text{ lb}
 \end{aligned}$$

Since the difference between the two determinations ( $Z$ ) is less than the repeatability limits for the two determinations ( $Y$ ), the two results are considered within the repeatability of the test method.

**X2.2 Reproducibility**—Single determinations of the Timken OK load are made by two laboratories on identical samples of Oil A. The laboratory results are Laboratory 1 = 40

lb and Laboratory 2 = 75 lb. Are these results within the reproducibility of the test method?

$$\begin{aligned}
 &\text{Laboratory 1} = 40 \text{ lb} \\
 &\text{Laboratory 2} = 75 \text{ lb} \\
 X &= \text{Average of the two results} = 58 \text{ lb} \\
 Y &= \text{Reproducibility limits for the two results} \\
 &= 74 \% \text{ of the mean value } X \\
 &= 0.74 \times 58 \text{ lb} \\
 &= 43 \text{ lb} \\
 Z &= \text{Difference between results} \\
 &= 75 \text{ lb} - 40 \text{ lb} \\
 &= 35 \text{ lb}
 \end{aligned}$$

Since the difference between the two determinations ( $Z$ ) is less than the reproducibility limits for the two determinations ( $Y$ ), the two results are considered within the reproducibility of the test method.

**X2.3** As additional examples of the precision limits, **Table X2.1** gives repeatability and reproducibility limits for selected average Timken OK loads. The difference between two determinations of given average OK load should exceed the values shown in **Table X2.1** only one in twenty times.

**TABLE X2.1 Repeatability and Reproducibility Limits**

NOTE 1—The values in **Table X2.1** are calculated as shown in the examples in **X2.1** and **X2.2**, and are not, therefore, necessarily multiples of 3 or 5 lb.

Average OK Load, lb	Repeatability, lb	Reproducibility, lb
15	4.5	11
30	9	22
40	12	30
50	15	37
60	18	44
70	21	52
80	24	59

### X3. UNIT PRESSURE

X3.1 Shown in **Table X3.1** are unit pressures in pounds per square inch. If determination of unit pressure is desired, measure the width of the wear scar after the 10-min run. Obtain

the contact pressure from **Table X3.1** by locating the wear scar width in inches for the corresponding load.

**TABLE X3.1 Unit Pressure Table for Timken Lubricant Tester**

Unit Pressure, psi = $\frac{10 \text{ (Load on Beam Arm, lb)}}{0.5 \text{ (Scar Width—in.)}}$																				
1 mm = 0.0394																				
Load on Beam Arm, lb	10	15	20	25	30	35	40	45	50	55	60	65	70	75	80	85	90	95	100	
Scar Width	Unit Pressure, lb/in. <sup>2</sup>																			
0.55 mm																				
0.0217 in.	9225	13 850	18 450	23 075	27 700	32 300	36 925	41 525	46 150	50 750	55 375	60 000	64 600	69 225	73 825	78 450	83 075	87 675	92 300	
0.60 mm																				
0.0236 in.	8475	12 700	16 925	21 150	25 375	29 600	33 850	38 075	42 300	46 525	50 750	55 000	59 225	63 450	67 675	71 900	76 150	80 375	84 600	
0.65 mm																				
0.0256 in.	7800	11 725	15 625	19 525	23 425	27 325	31 250	35 150	39 050	42 950	46 850	50 750	54 675	58 575	62 475	66 400	70 275	74 200	78 100	
0.70 mm																				
0.0276 in.	7250	10 875	14 500	18 125	21 750	25 375	29 000	32 625	36 250	39 875	43 500	47 125	50 750	54 375	58 025	61 650	65 275	68 900	72 525	
0.75 mm																				
0.0296 in.	6775	10 150	13 525	16 925	20 300	23 700	27 075	30 450	33 850	37 225	40 600	44 000	47 375	50 750	54 050	57 525	60 925	64 300	67 675	
0.80 mm																				
0.0315 in.	6350	9525	12 700	15 875	19 025	22 200	25 375	28 550	31 725	34 900	38 075	41 250	44 425	47 600	50 750	53 925	57 100	60 275	63 450	
0.85 mm																				
0.0335 in.	5975	8950	11 950	14 925	17 925	20 900	23 900	26 875	29 850	32 850	35 825	38 825	41 800	44 800	47 775	50 750	53 750	56 725	59 725	
0.90 mm																				
0.0355 in.	5650	8450	11 275	14 100	16 925	19 750	22 550	25 375	28 200	31 025	33 850	36 650	39 475	42 300	45 125	47 950	50 750	53 575	56 400	
0.95 mm																				
0.0374 in.	5350	8025	10 700	13 350	16 025	18 700	21 375	24 050	26 725	29 400	32 050	34 725	37 400	40 075	42 750	45 425	48 100	50 750	53 425	
1.00 mm																				
0.0394 in.	5075	7625	10 150	12 700	15 225	17 775	20 300	22 850	23 375	27 925	30 450	33 000	35 525	38 075	40 600	43 150	45 675	48 225	50 750	
1.05 mm																				
0.0414 in.	4825	7250	9675	12 075	14 500	16 925	19 350	21 750	24 175	26 600	29 000	31 425	33 850	36 250	38 675	41 100	43 500	45 925	48 350	
1.10 mm																				
0.0435 in.	4625	6925	9225	11 525	13 850	16 150	18 450	20 775	23 075	25 375	27 700	30 000	32 300	34 600	36 925	39 225	41 525	43 850	46 150	
1.15 mm																				
0.0453 in.	4425	6625	8825	11 025	13 250	16 450	17 650	19 875	22 075	24 275	26 475	28 700	30 900	33 100	35 300	37 525	39 725	41 925	44 150	
1.20 mm																				
0.0473 in.	4225	6350	8450	10 575	12 700	14 800	16 925	19 025	21 150	23 275	25 375	27 500	29 600	31 725	33 850	35 950	38 075	40 175	42 300	
1.25 mm																				
0.0493 in.	4050	6100	8125	10 180	12 175	14 225	16 250	18 275	20 300	22 325	24 375	26 400	28 426	30 450	32 475	34 525	36 550	38 575	40 600	
1.30 mm																				
0.0512 in.	3900	5850	7800	9750	11 725	13 675	15 625	17 575	19 525	21 475	23 425	25 400	27 325	29 275	31 250	33 200	35 150	37 100	39 050	
1.35 mm																				
0.0532 in.	3750	5650	7525	9400	11 275	13 150	15 050	16 925	18 800	20 675	22 550	24 450	26 325	28 200	30 075	31 950	33 850	35 725	37 600	
1.40 mm																				
0.0552 in.	3625	5450	7250	9075	10 875	12 700	14 500	15 325	18 125	19 950	21 750	23 575	25 375	27 200	29 000	30 825	32 625	34 450	36 250	
1.45 mm																				
0.0571 in.	3500	5250	7000	8750	10 500	12 250	14 000	15 750	17 500	19 250	21 000	22 750	24 500	26 250	28 000	29 750	31 500	33 250	35 000	
1.50 mm																				
0.0591 in.	3375	5075	6775	8450	10 150	11 850	13 525	15 225	16 925	18 625	20 300	22 000	23 700	25 375	27 075	28 775	30 450	32 150	33 850	
1.55 mm																				
0.0411 in.	3275	4900	6550	8175	9825	11 450	13 100	14 725	16 375	18 000	19 650	21 275	22 925	24 550	26 200	27 825	29 475	31 100	32 750	
1.60 mm																				
0.0631 in.	3175	4750	6350	7925	9525	11 100	12 700	14 275	15 875	17 450	19 025	20 625	22 200	23 800	25 375	26 950	28 550	30 150	31 725	
1.65 mm																				
0.0650 in.	3075	4625	6150	7700	9225	10 775	12 300	13 850	15 375	16 925	18 450	20 000	21 525	23 075	24 600	26 150	27 700	29 225	30 775	
1.70 mm																				
0.0670 in.	2975	4475	5975	7475	8950	10 450	11 950	13 425	14 925	16 425	17 925	19 400	20 900	22 400	23 900	25 375	26 875	28 375	29 850	
1.75 mm																				
0.0690 in.	2900	4350	5800	7250	8700	10 150	11 600	13 050	14 500	15 950	17 400	18 850	20 300	21 750	23 200	24 650	26 100	27 550	29 000	
1.80 mm																				
0.0709 in.	2825	4225	5650	7050	8450	9875	11 275	12 700	14 100	15 500	16 925	18 325	19 750	21 150	22 550	23 975	25 375	26 800	28 200	
1.85 mm																				
0.0729 in.	2750	4125	5500	6850	8225	9600	10 975	12 350	13 725	15 100	16 475	17 825	19 200	20 575	21 950	23 325	24 700	26 075	27 450	
1.90 mm																				
0.0749 in.	2675	4000	5350	6675	8025	9350	10 675	12 025	13 350	14 700	16 025	17 375	18 700	20 025	21 375	22 700	24 050	25 375	26 725	
1.95 mm																				
0.0768 in.	2600	3900	5200	6500	7800	9100	10 425	11 725	13 025	14 325	15 625	16 925	18 225	19 525	20 825	22 125	23 425	24 725	26 025	
2.00 mm																				
0.0788 in.	2550	3800	5075	6350	7625	8875	10 150	11 425	12 700	13 950	15 225	16 500	17 775	19 025	20 300	21 575	22 850	24 100	25 375	

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